# 7. Bearing Fits

# 7.1 Fitting

For rolling bearings, inner and outer rings are fixed on the shaft or in the housing so that relative movement does not occur between fitting surfaces during operation or under load. This relative movement between the fitting surfaces of the bearing and the shaft or housing can occur in a radial direction, an axial direction, or in the direction of rotation. Types of fitting include tight, transition and loose fitting, which may be selected depending on whether or not there is interference.

The most effective way to fix the fitting surfaces between a bearing's raceway and shaft or housing is to apply a "tight fit." The advantage of this tight fit for thin walled bearings is that it provides uniform load support over the entire ring circumference without any loss of load carrying capacity. However, with a tight fit, ease of installation and disassembly is lost; and when using a non-separable bearing as the floating-side bearing, axial displacement is not possible. For this reason, a tight fit cannot be recommended in all cases.

## 7.2 The necessity of a proper fit

In some cases, improper fit may lead to damage and shorten bearing life, therefore it is necessary to make a careful investigation in selecting a proper fit. Some of the bearing failure caused by improper fit are listed below.

- Raceway cracking, early flaking and displacement of raceway
- Raceway and shaft or housing abrasion caused by creeping and fretting corrosion
- Seizing caused by negative internal clearances

 Increased noise and deteriorated rotational accuracy due to raceway groove deformation

Please refer to insert pages A-96  $\sim$  A-99 for information concerning diagnosis of these conditions.

### 7.3 Fit selection

Selection of a proper fit is dependent upon thorough analysis of bearing operating conditions, including consideration of:

- Shaft and housing material, wall thickness, finished surface accuracy, etc.
- Machinery operating conditions (nature and magnitude of load, rotational speed, temperature, etc.)

## 7.3.1 "Tight fit" or "Loose fit"

(1) For raceways under rotating loads, a tight fit is necessary. (Refer to **Table 7.1**) "Raceways under rotating loads" refers to raceways receiving loads rotating relative to their radial direction. For raceways under static loads, on the other hand, a loose fit is sufficient.

(Example) Rotating inner ring load = the direction of the radial load on the inner ring is rotating relatively

(2) For non-separable bearings, such as deep groove ball bearings, it is generally recommended that either the inner ring or outer ring be given a loose fit.

Table 7.1	Radial	load	and	bearing	fit
-----------	--------	------	-----	---------	-----

Illustration	Bearing rotation	Ring load	Fit
Static load	Rota	r ring: ting Pr ring: onary Rotating inner ring load	Inner ring : Tight fit
Unbalanced load	Stati	r ring: onary ring load ring: ling	Outer ring : Loose fit
Static load	Stati	r ring: onary or ring: string: ring load	Inner ring : Loose fit
Unbalanced load	Rota	r ring: Rotating outer ring load er ring: onary	Outer ring : Tight fit



#### 7.3.2 Recommended Fits

Bearing fit is governed by the selection tolerances for bearing shaft diameters and housing bore diameters.

Widely used fits for 0 Class tolerance bearings and various shaft and housing bore diameter tolerances are shown in **Fig. 7.1**.

Generally-used, standard fits for most types of bearings and operating conditions are shown in **Tables 7.2 - 7.7**.

Table 7.2: Fits for radial bearings

Table 7.3: Fits for thrust bearings

Table 7.4: Fits for electric motor bearings

**Table 7.6**: Fits for inch series tapered roller bearings (ANSI Class 4)

**Table 7.7**: Fits for inch series tapered roller bearings (ANSI Class 3 and 0)

**Table 7.5**. shows fits and their numerical values. For special fits or applications, please consult **NTN** Engineering.

## 7.3.3 Interference minimum and maximum values

The following points should be considered when it is necessary to calculate the interference for an application:

- In calculating the minimum required amount of interference keep in mind that:
  - 1) interference is reduced by radial loads
  - 2) interference is reduced by differences between bearing temperature and ambient temperature
  - interference is reduced by variation of fitting surfaces
- The upper limit value should not exceed 1/1000 of the shaft diameter.

Required interference calculations are shown below.

## (1) Radial loads and required interference

Interference of the inner ring and shaft decreases when a radial load is applied to the bearing. The interference required to secure effective interference is expressed by formulae (7.1) and (7.2).

$$F_{r} \leq 0.3 C_{or}$$

$$\Delta_{dF} = 0.08 (d \cdot F_{r}/B)^{1/2}$$

$$= 0.25 (d \cdot F_{r}/B)^{1/2}$$

$$F_{r} > 0.3 C_{or}$$

$$\Delta_{dF} = 0.02 (F_{r}/B)$$

$$= 0.2 (F_{r}/B)$$

$$= 0.2 (F_{r}/B)$$

$$\{kgf\}\}$$

$$\{kgf\}\}$$
.....(7.1)

Where,

 $\Delta_{\it dF}$ : Required effective interference according to radial load  $\,\mu$  m

 $d\,$  : Bearing bore diameter mm

B: Inner ring width mm  $F_r$ : Radial load N {kgf}

 $C_{or}$ : Basic static load rating N {kgf}

### (2) Temperature difference and required interference

Interference between inner rings and steel shafts is reduced as a result of temperature increases (difference between bearing temperature and ambient temperature,  $\Delta T)$  caused by bearing rotation. Calculation of the minimum required amount of interference in such cases is

shown in formula (7.3).

 $\Delta_{dT} = 0.0015 \cdot d \cdot \Delta T$  .....(7.3)

 $\Delta_{dT}$ : Required effective interference for temperature difference  $\mu$  m

ΔT: Difference between bearing temperature and ambient temperature °C

d: Bearing bore diameter mm

## (3) Fitting surface variation and required interference

Interference decreases because the fitting surface is smoothened by fitting (surface roughness is reduced). The amount the interference decreases depends on roughness of the fitting surface. It is generally necessary to anticipate the following decrease in interference.

For ground shafts:  $1.0\sim2.5 \mu$  m For lathed shafts:  $5.0\sim7.0 \mu$  m

## (4) Maximum interference

When bearing rings are installed with an interference fit, tension or compression stress may occur along their raceways. If interference is too great, this may cause damage to the rings and reduce bearing life. You should try to obtain the previously described upper limit.

## 7.3.4 Other details

- (1) Tight interference fits are recommended for,
  - Operating conditions with large vibration or shock loads
  - Applications using hollow shafts or housings with thin walls
  - Applications using housings made of light alloys or plastic
- (2) Small interference fits are preferable for,
  - Applications requiring high running accuracy
  - Applications using small sized bearings or thin walled bearings
- (3) Consideration must also be given to the fact that fit selection will effect internal bearing clearance selection. (refer to page insert A-58)

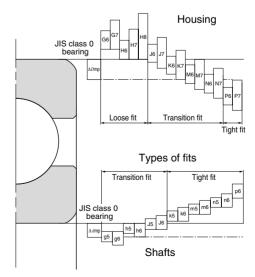


Fig 7.1 State of fitting

(4) A particular type of fit is recommended for SL type cylindrical roller bearings.

Table 7.2 General standards for radial bearing fits (JIS Class 0, 6X, 6)

Table 7.2 (1) Tolerance class of shafts commonly used for radial bearings (classes 0, 6X and 6)

•	Conditions		arings		ylindrical roller bearing Tapered roller bearing		oller bearing	Shaft	Remarks
C	onaitions			Shaft dian	neter (mm)			tolerance class	Hemarks
		Over	Under	Over	Under	Over	Under		
				Cylindrical	bore bearing	g (Classes 0	, 6X and 6)		
Inner rin of ur	Light load or fluctuating load	18 100 —	18 100 200	40 140	40 140 200	_ _ _ _		h5 js6 k6 m6	When greater accuracy is required js5, k5, and m5 may be substituted for js6, k6, and m6.
Inner ring rotational load or load of undetermined direction	Ordinary <sup>●</sup> load	18 100 140 200	18 100 140 200 280	40 100 140 200	40 100 140 200 400	40 65 100 140 280	40 65 100 140 280 500	js5 k5 m5 m6 n6 p6 r6	Alteration of inner clearances to accommodate fit is not a consideration with single-row angular contact bearings and tapered roller bearings. Therefore, k5 and m5 may be substituted for k6 and m6.
or load ion	Heavy load or impact load	_ _ _	_ _ _	50 140 200	140 200	50 100 140	100 140 200	n6 p6 r6	Use bearings with larger internal clearances than CN clearance bearings.
Inner ring static load	Inner ring must move easily over shaft			Overall sha	aft diameter			g6	When greater accuracy is required use g5. For large bearings, f6 will suffice for to facilitate movement.
ring	Inner does not have to move easily over shaft			Overall sha	aft diameter		h6	When greater accuracy is required use h5.	
Cen	Center axial load Overall shaft diameter							js6	Generally, shaft and inner rings are not fixed using interferance.
			Tapered	bore bearing	g (class 0) (v	vith adapter	or withdrawa	al sleeve)	
0	verall load			Overall sha	aft diameter			h9/IT5	h10/IT7  will suffice for power transmitting shafts.

Table 7.2 (2) Fit with shaft (fits for tapered bore bearings (Class 0) with adapter assembly/withdrawal sleeve)

All loads	All bearing types	All shaft diameters	Tolerance	h9 / IT5 <b>2</b>	General applications
All loads	All bearing types		class	h10/ IT7 2	Transmission shafts, etc.

1 Standards for light loads, normal loads, and heavy loads

 $\begin{cases} \text{Light loads: equivalent radial load} \leq 0.06 \ \mathit{Cr} \\ \text{Normal loads: } 0.06 \ \mathit{Cr} < \text{equivalent radial load} \leq 0.12 \ \mathit{Cr} \end{cases}$ 

Heavy loads:  $0.12 C_r < equivalent radial load$ 

2 IT5 and IT7 show shaft roundness tolerances, cylindricity tolerances, and related values.

Note: All values and fits listed in the above tables are for solid steel shafts.



Table 7.2 (3) Tolerance class of housing bore commonly used for radial bearings (classes 0, 6X and 6)

		Conditions		Toleration class	
Housing	Туре	s of load	Outer ring axial <sup>②</sup> direction movement	of housing bore	Remarks
Single housing or divided housing		All types of loads	Able to move.	Н7	G7 will suffice for large bearings or bearings with large temperature differential between the outer ring and housing.
		Light load <sup>●</sup> or ordinary load <sup>●</sup>	Able to move.	H8	_
	Outer ring static load	Shaft and inner ring become hot.	Able to move easily.	<b>G</b> 7	F7 will suffice for large bearings or bearings with large temperature differential between the outer ring and housing.
		Requires precision rotation with light	As a rule, cannot move.	K6	Primarily applies to roller bearings.
		or ordinary loads.	Able to move.	JS6	Primarily applies to ball bearings.
		Requires quiet operation.	Able to move.	H6	
		Light or ordinary load	Able to move.	JS7	If precision is required, JS6 and K6 are used in place of
Single housing	Indeterminate load	Ordinary load or heavy load •	As a rule, cannot move.	K7	JS7 and K7.
		Large impact load	Cannot move.	M7	
		Light or fluctuating load	Cannot move.	M7	_
	Outer ring	Ordinary or heavy load	Cannot move.	N7	Primarily applies to ball bearings.
	rotational load	Heavy load or large impact load with thin housing	Cannot move.	P7	Primarily applies to roller bearings.

1 Standards for light loads, normal loads, and heavy loads

Light loads: equivalent radial load  $\leq 0.06 C_r$ 

Normal loads:  $0.06 C_r < \text{equivalent radial load} \le 0.12 C_r$ 

Heavy loads: 0.12  $C_{\rm r}$  < equivalent radial load

2 Indicates whether or not outer ring axial displacement is possible with non-separable type bearings.

Note 1: All values and fits listed in the above tables are for cast iron or steel housings.

2: If only center axial load is applied to the bearing, select a tolerance class that provides clearance for the outer ring in the axial direction.



# Table 7.3 Standard fits for thrust bearings (JIS Class 0 and 6)

Table 7.3 (1) Shaft fits

Bearing type		Load conditions	Fit	Shaft diameter mm over incl.	Tolerance class
All thrust bearings		Centered axial load only	Transition fit	All sizes	js6 or h6
	Cor	Inner ring static load	Transition fit	All sizes	js6
Spherical roller thrust bearings	Inner ring static load  Inner ring rotating load  or  Indeterminate load		Transition fit Tight fit	— ~ 200 200 ~ 400 400 ~	k6 or js6 m6 or k6 n6 or m6

Table 7.3 (2) Housing fits

	Bearing type		Load conditions	Fit	Tolerance class	Remarks
	All thrust bearings	C 0.	ntered axial load only		Select a tolerance	class that will provide clearance between outer ring and housing.
		Cei	mered axiai load only	Loose fit	H8	Greater accuracy required with thrust ball bearings
ĺ	Spherical roller thrust bearings	Com	Outer ring static load		H7	<u>—</u>
		bined	Indeterminate	Transition fit	K7	Normal operating conditions
		load	load or outer ring rotating load		M7	For relatively large radial loads

Note: All values and fits listed in the above tables are for cast iron or steel housings.

Table 7.4 Fits for electric motor bearings

		ft fits	Housing fits		
Bearing type	Shaft diameter mm over incl.	Tolerance class	Housing bore diameter	Tolerance class	
Deep groove ball bearings	~ 18 18 ~100 100 ~160	j5 k5 m5	All sizes	H6 or J6	
Cylindrical roller bearings	~ 40 40 ~160 160 ~200	k5 m5 n6	All sizes	H6 or J6	

Table 7.5 Numeric value table of fitting for radial bearing of 0 class

Table 7.5 (1) Fitting against shaft

Table 7.5 (1) Fitting against shart												
Nomina diame			n bore •	g5	g6	h5	h6	j5	js5	j6		
bear			ation	bearing shaft								
d		$\Delta_c$	<i>i</i> mp									
mı	m											
over	incl.	high	low									
3	6	0	-8	4T∼ 9L	4T~12L	8T∼ 5L	8T∼ 8L	11T~ 2L	10.5T~ 2.5L	14T∼ 2L		
6	10	0	-8	3T∼11L	3T∼14L	8T∼ 6L	8T∼ 9L	12T∼ 2L	11T ~ 3L	15T∼ 2L		
10	18	0	-8	2T~14L	2T~17L	8T∼ 8L	8T~11L	13T∼ 3L	12T ~ 4L	16T∼ 3L		
18	30	0	-10	3T~16L	3T~20L	10T∼ 9L	10T~13L	15T∼ 4L	14.5T~ 4.5L	19T∼ 4L		
30	50	0	-12	3T~20L	3T~25L	12T~11L	12T~16L	18T∼ 5L	17.5T~ 5.5L	23T~ 5L		
50	80	0	-15	5T~23L	5T~29L	15T∼13L	15T~19L	21T~ 7L	21.5T~ 6.5L	27T~ 7L		
80	120	0	-20	8T∼27L	8T∼34L	20T~15L	20T~22L	26T~ 9L	27.5T~ 7.5L	33T∼ 9L		
120 140 160	140 160 180	0	-25	11T~32L	11T∼39L	25T~ 18L	25T~25L	32T~11L	34T ∼ 9L	39T~11L		
180 200 225	200 225 250	0	-30	15T~35L	15T~44L	30T∼20L	30T∼29L	37T~13L	40T ∼10L	46T∼13L		
250 280	280 315	0	-35	18T~40L	18T~49L	35T~23L	35T~32L	42T~16L	46.5T~11.5L	51T~16L		
315 355	355 400	0	-40	22T~43L	22T~54L	40T∼25L	40T∼36L	47T~18L	52.5T~12.5L	58T∼18L		
400 450	450 500	0	-45	25T~47L	25T~60L	45T∼27L	45T~40L	52T~20L	58.5T~13.5L	65T~20L		

lacktriangle Above table is not applicable to tapered roller bearings whose bore diameter d is 30mm or less.

Table 7.5 (2) Fitting against housing

Nom		Mean o		G7	H6	H7	J6	J7	Js7	K6
outside diameter of bearing		diameter deviation		housing bearing						
D <b>mm</b>		$\Delta D$ mp					#			
over	incl.	high	low							
6	10	0	-8	5L∼ 28L	0∼17L	0∼ 23L	4T~13L	7T~16L	7.5T~15.5L	7T~10L
10	18	0	-8	6L∼ 32L	0∼19L	0∼ 26L	5T~14L	8T∼18L	9T ∼17L	9T~10L
18	30	0	-9	7L∼ 37L	0~22L	0∼ 30L	5T~17L	9T∼21L	10.5T~19.5L	11T~11L
30	50	0	-11	9L∼ 45L	0∼27L	0∼ 36L	6T~21L	11T~25L	12.5T~23.5L	13T~14L
50	80	0	-13	10L∼ 53L	0∼32L	0∼ 43L	6T~26L	12T~31L	15T ∼28L	15T~17L
80	120	0	-15	12L~ 62L	0∼37L	0∼ 50L	6T~31L	13T∼37L	17.5T~32.5L	18T~19L
120	150	0	-18	14L~ 72L	0~43L	0∼ 58L	7T~36L	14T~44L	20T ∼38L	21T~22L
150	180	0	-25	14L~ 79L	0~50L	0∼ 65L	7T~43L	14T~51L	20T ~45L	21T~29L
180	250	0	-30	15L~ 91L	0~59L	0∼ 76L	7T~52L	16T~60L	23T ~53L	24T~35L
250	315	0	-35	17L~104L	0~67L	0∼ 87L	7T~60L	16T~71L	26T ~61L	27T~40L
315	400	0	-40	18L~115L	0~76L	0∼ 97L	7T~69L	18T~79L	28.5T~68.5L	29T~47L
400	500	0	-45	20L~128L	0∼85L	0~108L	7T~78L	20T~88L	31.5T~76.5L	32T~53L

② Above table is not applicable to tapered roller bearings whose outside diameter D is 150mm or less. Note: Fitting symbol "L" indicates clearance and "T" indicates interference.



	Init	"	m
_		m	

									nit μ m
js6	k5	k6	m5	m6	n6	p6	r6	Nominal diamete	
bearing shaft	bearing shaft	bearing shaft	bearing shaft	bearin					
_	#			#				d mm	
								over	incl.
12T ~ 4L	14T~1T	17T~1T	17T∼ 4T	20T~ 4T	24T~ 8T	28T~12T		3	6
12.5T∼ 4.5L	15T~1T	18T~1T	20T∼ 6T	23T∼ 6T	27T~10T	32T~15T		6	10
13.5T∼ 5.5L	17T~1T	20T~1T	23T~ 7T	26T~ 7T	31T~12T	37T~18T		10	18
16.5T∼ 6.5L	21T~2T	25T~2T	27T~ 8T	31T∼ 8T	38T~15T	45T~22T		18	30
20T ~ 8L	25T~2T	30T~2T	32T~ 9T	37T∼ 9T	45T~17T	54T~26T		30	50
24.5T~ 9.5L	30T∼2T	36T~2T	39T~11T	45T~11T	54T~20T	66T~32T		50	80
31T ~11L	38T~3T	45T~2T	48T~13T	55T~13T 65T~23T		79T~37T		80 1	120
37.5T~12.5L	46T∼3T	53T∼3T	58T~15T	65T~15T	77T~27T	93T~43T	113T~ 63T 115T~ 65T 118T~ 68T	140 1	140 160 180
44.5T~14.5L	54T~4T	63T∼4T	67T~17T	76T~17T	90T~31T	109T~50T	136T~ 77T 139T~ 80T 143T~ 84T	200 2	200 225 250
51T ~16L	62T~4T	71T~4T	78T~20T	87T~20T	101T~34T	123T~56T	161T~ 94T 165T~ 98T		280 315
58T ~18L	69T~4T	80T~4T	86T~21T	97T~21T	113T~37T	138T~62T	184T~108T 190T~114T		355 400
65T ~20L	77T~5T	90T~4T	95T~23T	108T~23T	125T~40T	153T~68T	211T~126T 217T~132T		450 500

K7	M7	N7	P7	Nominal outside
housing bearing	housing bearing	housing bearing	housing bearing	diameter of bearing
#	#			D mm
				over incl.
10T~13L	15T∼ 8L	19T∼ 4L	24T~ 1T	6 10
12T~14L	18T∼ 8L	23T~ 3L	29T~ 3T	10 18
15T~15L	21T~ 9L	28T~ 2L	35T∼ 5T	18 30
18T~18L	25T~11L	33T∼ 3L	42T∼ 6T	30 50
21T~22L	30T~13L	39T∼ 4L	51T∼ 8T	50 80
25T~25L	35T~15L	45T∼ 5L	59T∼ 9T	80 120
28T~30L	40T∼18L	52T∼ 6L	68T~10T	120 150
28T~37L	40T∼25L	52T~13L	68T∼ 3T	150 180
33T~43L	46T∼30L	60T~16L	79T∼ 3T	180 250
36T~51L	52T~35L	66T~21L	88T∼ 1T	250 315
40T~57L	57T~40L	73T~24L	98T∼ 1T	315 400
45T~63L	63T~45L	80T~28L	108T∼ 0	400 500



Table 7.6 General fitting standards for tapered roller bearings using US customary unit (ANSI class 4)

Table 7.6 (1) Fit with shaft

Unit µm

	Onit µm									
Operating conditions		Nominal bearing bore diameter $d$ mm over incl.	Bore diameter tolerance  \$\Delta_{ds}\$ high low		Shaft diameter tolerance		Fitting • max min		Remark	
		OVCI IIIOI.	riigii	IOVV	riigii	1044	max	111111		
=		~ 76.2	+13	0	+ 38	+ 25	38T ∼	12T	Applicable when	
iner	0.001500.000.100.001	76.2 ~ 304.8	+25	0	+ 64	+ 38	64T ~	13T	slight impact	
₹.	Ordinary load	304.8 ~ 609.6	+51	0	+127	+ 76	127T ~	25T	load is applied	
Inner ring rotational load		609.6 ~ 914.4	+76	0	+190	+114	190T ~	38T	as well.	
tatio	Heavy load Impact load	~ 76.2	+13	0	+ 64	+ 38	38T ~	12T		
ona		76.2 ~ 304.8	+25	0	0.5 μm	n mean inter	rference per 1 mm of inner ring bore			
<u></u>		304.8 ~ 609.6	+51	0	diameter. Minimum interference is 25 $\mu$ m. Tolerance for the shaft is adjusted to match tolerance of bearing bore diameters.					
g		609.6 ~ 914.4	+76	0						
0	Inner ring does	~ 76.2	+13	0	+ 13	0	13T ~	13L		
ute	not have to	76.2 ~ 304.8	+25	0	+ 25	0	25T ~	25L		
₹.	move easily over shaft with	304.8 ~ 609.6	+51	0	+ 51	0	51T ~	51L	Not applicable	
og ro	ordinary load.	609.6 ~ 914.4	+76	0	+ 76	0	76T ~	76L	when impact	
Outer ring rotational load	Inner ring must move easily over shaft with ordinary load.	~ 76.2	+13	0	0	- 13	0 ~	13L	load is applied.	
ona		76.2 ~ 304.8	+25	0	0	- 25	0 ~	50L		
<u></u>		304.8 ~ 609.6	+51	0	0	- 51	0 ~	102L		
ad		609.6 ~ 914.4	+76	0	0	- 76	0 ~	152L		

Table 7.6 (2) Fit with housing

Unit  $\mu$  m

Gin p									J 2
Operating conditions		Nominal bearing outer diameter  D mm over incl.	Outer diameter tolerance $\Delta_{Ds}$ high low		Housing bore diameter tolerance high low		Fitting fritting fritting fritting		Types of fit
		OVCI IIICI.	mgm	IOW	riigii	IOW	max	111111	
		~ 76.2	+25	0	+ 76	+ 51	26L ~	- 76L	
	When used on	76.2 ~ 127.0	+25	0	+ 76	+ 51	26L ~	- 76L	
	floating- or	127.0 ~ 304.8	+25	0	+ 76	+ 51	26L ~	- 76L	loose fit
	fixed side	304.8 ~ 609.6	+51	0	+152	+102	51L ~	- 152L	
Inner ring		609.6 ~ 914.4	+76	0	+229	+152	76L ~	229L	
er m.	When outer ring is adjusted in axial direction	~ 76.2	+25	0	+ 25	0	25T ~	- 25L	
ng		76.2 ~ 127.0	+25	0	+ 25	0	25T ~	25L	transition fit
ot		127.0 ~ 304.8	+25	0	+ 51	0	25T ~	- 51L	
atic		304.8 ~ 609.6	+51	0	+ 76	+ 26	25T ~	- 76L	
rotational load		609.6 ~ 914.4	+76	0	+127	+ 51	25T ~	-	
00	When outer ring is not adjusted in axial direction	~ 76.2	+25	0	- 13	- 38	63T ~	- 13T	
٥		76.2 ~ 127.0	+25	0	- 25	- 51	76T ~	25T	
		127.0 ~ 304.8	+25	0	- 25	- 51	76T ~	25T	
		304.8 ~ 609.6	+51	0	- 25	- 76	127T ~	25T	
		609.6 ~ 914.4	+76	0	- 25	-102	178T ~	_	
		~ 76.2	+25	0	- 13	- 38	63T ~	- 13T	tight fit
<u> </u>	When outer	76.2 ~ 127.0	+25	0	- 25	- 51	76T ~	-	
Outer ring rotational load	ring is not	127.0 ~ 304.8	+25	0	- 25	- 51	76T ~		
<u>a</u> =	adjusted in axial direction	304.8 ~ 609.6	+51	0	- 25	- 76	127T ~	_	
oa	axiai uii eciiofi	609.6 ~ 914.4	+76	0	- 25	-102	178T ~		
<u>o</u>		003.0 314.4	T/0	U	- 23	-102	1701	231	

<sup>•</sup> Fitting symbol "L" indicates clearance and "T" indicates interference.



Table 7.7 General fitting standards for tapered roller bearings using US customary unit (ANSI classes 3 and 0)

Table 7. (1) Fit with shaft

Unit  $\mu$  m

Operating conditions		Nominal bearing bore diameter $d \mod$	Bore diameter tolerance		Shaft diameter tolerance		Fitting		
		over incl.	high	low	high	low	max min		
п	Precision	~ 304.8	+13	0	+ 30	+ 18	30T ∼ 5T		
Inner ring	machine tool spindles	304.8 ~ 609.6	+25	0	+ 64	+ 38	64T ∼ 13T		
ing		609.6 ~ 914.4	+38	0	+102	+ 64	102T ∼ 26T		
rotational load	Heavy load	~ 76.2	+13	0	Minimum interference is 0.25 $\mu$ m per 1 mm of inner ring bore diameter.				
tion	Impact load High-speed rotation	76.2 ~ 304.8	+13	0					
<u>a</u>		304.8 ~ 609.6	+25	0					
_		609.6 ~ 914.4	+38	0					
Outer ring rotational load		~ 304.8	+13	0	+ 13	0	30T ∼ 5T		
ute	Precision machine tool spindles		_		_	0			
a ri		304.8 ~ 609.6	+25	0	+ 25	-	64T ~ 13T		
ng oad	opinaios	609.6 ~ 914.4	+38	0	+102	0	102T ~ 26T		

Note: For class 0, bearing bore diameter d applies to 241.3 mm or less.

Table 7.7 (2) Fit with housing

Unit  $\mu$  m

									Offit # III
Operating conditions outer diag			Outer di tolera	ince Os	Housing bore diameter tolerance		Fitting •		Type of fit
		over incl.	high	low	high	low	max	min	
		~ 152.4	+13	0	+ 38	+ 25	12L ~	38L	
	) A/I	152.4 ~ 304.8	+13	0	+ 38	+ 25	12L ~	38L	
	When used for floating-side	304.8 ~ 609.6	+25	0	+ 64	+ 38	13L ~	64L	
	lioating side	609.6 ~ 914.4	+38	0	+ 89	+ 51	13L ~	89L	1
<del>_</del>		~ 152.4	+13	0	+ 25	+ 13	0 ~	25L	loose fit
Inner ring	When used for fixed side	152.4 ~ 304.8	+13	0	+ 25	+ 13	0 ~	25L	
⊒.		304.8 ~ 609.6	+25	0	+ 51	+ 25	0 ~	51L	
g ro		609.6 ~ 914.4	+38	0	+ 76	+ 38	0 ~	76L	
rotational load	When outer ring is adjusted in axial direction	~ 152.4	+13	0	+ 13	0	13T ~	13L	transition fit
on O		152.4 ~ 304.8	+13	0	+ 13	0	13T ∼	13L	
<u>a</u>		304.8 ~ 609.6	+13	0	+ 25	0	25T ~	25L	
ad		609.6 ~ 914.4	+38	0	+ 38	0	38T ∼	38L	
	When outer ring is not adjusted in axial direction	~ 152.4	+13	0	0	- 13	26T ~	0	
		152.4 ~ 304.8	+13	0	0	- 25	38T ∼	0	
		304.8 ~ 609.6	+25	0	0	- 25	50T ∼	0	
		609.6 ~ 914.4	+38	0	0	- 38	76T ∼	0	
<u> </u>	Ordinary load	~ 152.4	+13	0	- 13	- 25	38T ~	13T	tight fit
Oute	When outer ring	152.4 ~ 304.8	+13	0	- 13	- 38	51T ∼	13T	
nal I	is not adjusted	304.8 ~ 609.6	+25	0	- 13	- 38	63T ∼	13T	
Ordinary lo When oute is not adjust in axial dire	in axial direction	609.6 ~ 914.4	+38	0	- 13	- 51	89T ∼	13T	

lacktriangle Fitting symbol "L" indicates clearance and "T" indicates interference. Note: For class 0, bearing outer diameter D applies to 304.8 mm or less.